

# Design and simulation of an optimized electrothermal microactuator with Z-shaped beams

## Diseño y simulación de un microactuador electrotérmico optimizado con brazos en forma Z

Margarita Tecpoyotl-Torres\*, Ramón Cabello-Ruiz\*, José Gerardo Vera-Dimas\*

### ABSTRACT

The displacement of the central shuttle of a Z-shape chevron actuator can be calculated using a developed approach from other authors. Who demonstrated that the actuators with this geometry offer a larger displacement compared with V-shape actuators. Z-shape offers a larger stiffness and output force for the case of only one arm. This paper is focused on the optimization of the Z-shaped beams of a chevron actuator of eight beams, which seeks to increase the previously described response. The structure is designed in parametric solid modeling 3D software Autodesk Inventor, and simulated by finite element method in Ansys 15.0. These simulations were implemented considering several modifications on the length of the Z-shaped beams in order to choose the most appropriate length. The electric potential applied in all cases was from 0.2 V up to 5 V. The Z-shape length of the arms for the case of the optimized Z-shape actuator increases the shuttle's displacement in approximately 50% compared to V-shape actuator, and 38% compare to the original Z-shape. Analytical adjusted approach is extremely matched with the simulations results. Length of the Z-shape beam is the determinant factor of the displacement. The low stiffness of the optimized Z-shape actuator (89% lower than the original V-shape and 58% compared to Z-shape) can allow its use as load sensor.

### RESUMEN

El desplazamiento de la flecha central de un actuador chevrón con brazos basados en una geometría de forma Z puede ser calculado utilizando una aproximación desarrollada por otros autores, quienes probaron que los actuadores con un solo brazo ofrecen un mayor desplazamiento comparado con los actuadores en forma V convencionales, y que a su vez también ofrecen mayor rango de rigidez y fuerza de salida. Este artículo está enfocado en optimizar un actuador de ocho brazos en forma Z, buscando incrementar la respuesta anteriormente mencionada. La estructura fue diseñada en un paquete de modelado paramétrico de sólidos en 3D, Autodesk Inventor y simulada por el método de elemento finito en Ansys 15.0. Se implementaron varias longitudes en el largo de la forma Z, con la finalidad de seleccionar la longitud que proporcione el mayor desplazamiento. El potencial eléctrico aplicado en todos los casos fue de 0.2 V hasta 5 V. En los resultados se muestra que utilizando la longitud de la forma Z de los brazos del actuador Z optimizado se obtiene un incremento en el desplazamiento de la flecha de aproximadamente 50% comparado con el obtenido en el actuador V, y de un 38% en el actuador Z original. Además, se puede notar que estos resultados, empleando la aproximación analítica, son extremadamente cercanos a los resultados obtenidos mediante la simulación. La longitud total de los brazos con forma Z es el factor determinante del desplazamiento de la flecha. La baja rigidez mostrada por el actuador Z optimizado (89% menor que el actuador V y 58% comparado con el chevrón Z original) hace que pueda utilizarse como sensor de carga.

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MEMS; chevron; electrothermal; displacement.

#### Palabras clave:

MEMS; chevrón; electrotérmico; desplazamiento.

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### INTRODUCTION

The Micro Electro Mechanical System (MEMS) consists of mechanical structures, microsensors, microactuators and microelectronic devices. Microactuators convert electrical input to mechanical output. Vertical actuators

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are useful for many applications requiring out-of-plane displacements. Such applications include tunable parallel plate capacitors and other passive components, optical modulators, RF and optical switches, micromanipulators, devices inside of Scanning Electron Microscopes (SEM) (Cao, Chen, Filleter & Sun, 2015); (Varona, Tecpoyotl-Torres & Hamoui, 2009). It has been demonstrated that electrothermal actuators are compact, stable, and high-force devices under low actuation voltages (Chow & Lai, 2009). Actuation is based on the thermal expansion caused by Joule heating. Usually, the heating is achieved by passing current through the beams to cause resistive heating. V-shaped actuator or chevron is widely used due to its small footprint and high force (on the order of mN) (Zhu & Espinosa, 2005; Zhu, Moheimani & Yuce, 2012), but they cannot achieve large motion (typically up to a few  $\mu\text{m}$ ) (Zhu & Espinosa, 2005). Two possible limitations exist for the V-shaped actuators. The first limitation comes from the slanted beams. Slanted beams pose challenges for fabricating small features with smooth sidewall surfaces, which deteriorate as the beams width gets close to the resolution of photolithography (typically  $\sim 2 \mu\text{m}$ ). Other limitation is due to the large stiffness of the V-shaped actuator (on the order of thousands of  $\text{N m}^{-1}$  and above). As a result, the V-shaped actuators cannot be used as load sensors and actuators simultaneously (Guan & Zhu, 2010). Thus, additional load sensors are required for applications such as nanomechanical testing (Zhu & Espinoza, 2005) and nanomanufacturing (Dong & Ferreira, 2008). Z-shape actuator unlike the previous, facilitates the fabrication due to not presents the slanted beams. Unique angles involves are present in Z-shape and they are right angles (Guan & Zhu, 2010). While the basic principle of operation is similar to the conventional actuator with V-shaped beams, the configuration of the Z-shaped beams means that the mechanism's mechanical stiffness in the reverse direction is similar to that in the direction of actuation. Thus, two actuators can be effectively coupled back-to-back to create a structure that possesses bidirectional motion (Rakotondrabe, Fowler & Moheimani, 2014).

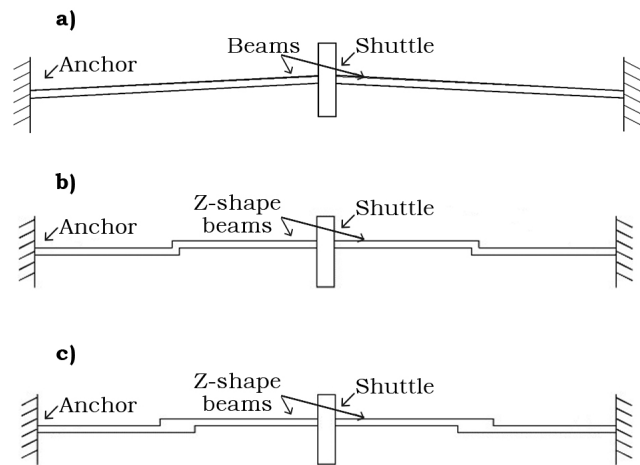
The objective of this paper is to present an electrothermal microactuator optimizing the length of the Z-shaped beams based on the geometry presented in (Guan & Zhu, 2010), which provides a larger displacement, compared to the V-shape actuator. An analysis of the beam shape was realized to find the length providing the larger displacements of the central shuttle. Simulations require the application of an electrical potential in the anchors of the actuator to thereby obtain a displacement caused by the heating produced in the

beams by the Joule effect. For the optimized structure, a comparative graph between the displacements obtained from both analytical and simulated calculations are presented.

## MATERIALS AND METHODS

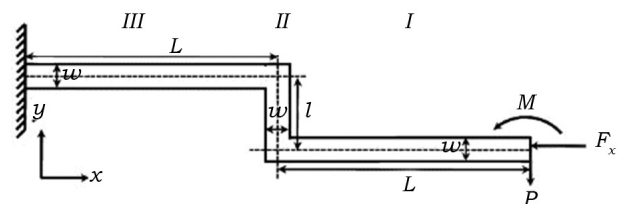
Figure 1, shows the schematic of the V-shaped, Z-shaped and proposed Z-shaped electrothermal actuators in order to compare them. Their elements are anchors, beams and shuttle.

The thermomechanical response was analytically derived on the base of the following assumptions: the central shuttle is rigid and its thermal expansion is neglected, figure 2 (Guan & Zhu, 2010; Zhu, Corigliano & Espinoza, 2006), where the authors divided the total length in three sections in order to obtain the displacement approximation.



**Figure 1.** a) Schematic of the V-shaped thermal actuator. b) Schematic of the Z-shaped thermal actuator (Guan & Zhu, 2010). c) Proposed design of Z-shaped thermal actuator.

Source: Figures a) and b) author's own elaboration based on Guan & Zhu (2010), figure c) is author's own elaboration & proposal.



**Figure 2.** Free-body diagram of a single Z-shaped beam.

Source: Guan & Zhu (2010).

The displacement in the shuttle can be obtained from equation (1) (Guan & Zhu, 2010):

$$U_y = \frac{12\alpha\Delta TL^3}{l^2 + 6L\left(1 + \frac{w^2}{3l}\right)}, \quad (1)$$

where:  $U_y$  = displacement in y direction,  $\alpha$  = coefficient of thermal expansion,  $\Delta T$  = temperature difference,  $L$  = length of the long arm beam,  $l$  = length of the central beam, and  $w$  = beam width.

The output force is given by the product of displacement and stiffness as it was established in (Guan & Zhu, 2010), where the cases of one and two beams are considered in their analyses (2):

$$f = kU. \quad (2)$$

Equation (2) can be generalized, considering  $n$  beams:

$$f = nkU. \quad (3)$$

In order to remain a symmetrical geometry in the entire arm, the lateral width of the arm, at the opposite endings of the Z-shape, can be considered as:

$$w_{z\text{-modified}} = w_{z\text{-original}} * 2. \quad (4)$$

Where:  $w_{z\text{-modified}}$  corresponds to the Z-shape width.

The stiffness  $k$  is provided by equation (5) (Guan & Zhu, 2010):

$$k = \frac{Ew^3h(l^3 + 2Lw^2 + 6Ll^2)}{(8L^3l^3 + w^2l^4 + 16w^2L^4 + 2w^4Ll + 12L^4l^2 + 6w^2Ll^3)}. \quad (5)$$

Where:  $E$  = Young's modulus, and  $h$  = beam thickness.

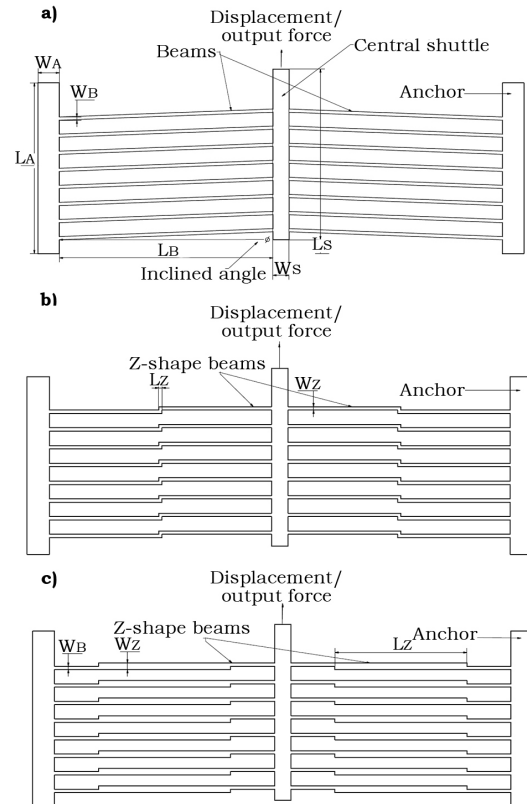
In this work, the arm modification is given using multiples of  $w$  from 1  $w$  up to 66  $w$  for the Z-shape length. The total length of the beam remains constant at 2  $L$ .

## RESULTS AND DISCUSSION

The V-shape actuator is based on the horizontal actuator originally proposed in (Cragun & Howell, 1999). In general, this electrothermal actuator uses an array of beams facing each other in pairs to generate one-directional displacement. Heating of the beam-pairs causes them to expand and ultimately buckle. The beams are designed with a pre-bend angle  $\alpha$ , so buckling has a ten-

dency to move in-plane (parallel to the substrate) as it is depicted in figure 3. The basic principle of actuation is the same as the case of V-shape actuator.

The structures were made of silicon, whose properties are shown in table 1.



**Figure 3.** a) V-shaped actuator (Varona, Tecpoyotl-Torres & Hamoui 2009). b) Z-shape actuator. c) Proposed Z-shape.

Source: Figures a) author's own elaboration based on Varona *et al.* (2009), figures b) and c) are author's own elaboration & proposal.

**Table 1.**  
Silicon properties.

Property	Value
Density ( $\rho$ ) in Kg/m <sup>3</sup>	2330
Coefficient of thermal expansion ( $\alpha$ ) in 1/°C	$2.6 \times 10^{-6}$
Conductivity ( $k$ ) in W/m°C	124
Heat Specific ( $C_p$ ) in J/Kg°C	702
Young's Modulus ( $E$ ) in GPa	131
Poisson's ratio ( $\mu$ )	0.33
Resistivity ( $\rho$ ) in $\Omega \cdot m$	$1 \times 10^{-4}$
Permeability ( $\mu$ )	1

Source: Callón (N. D.).

From the obtained results, it can be observed that the proposed Z-shape actuator with  $l_z$  equal to  $40w_B$  shows the largest displacement value on the considered voltage range applied. Initially, a linear response can be observed until the maximum point at  $40w_B$ , after it, the curved has a considerable drop. The optimized length of the Z-shape actuator, which provides the largest displacement corresponds to  $l_z$  equal to  $40w_B$  (figure 4).

The dimensions used are presented in table 2.

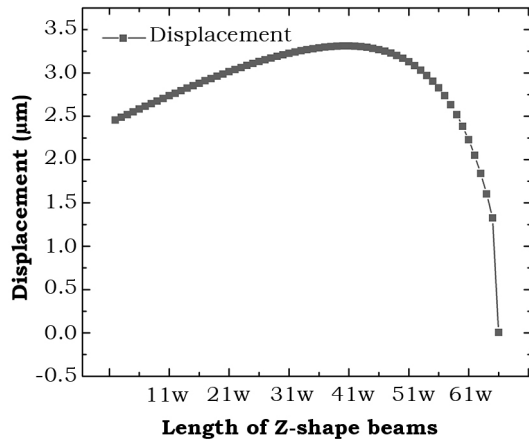


Figure 4. Displacement obtained for each Z-shape arm length. Source: Author's own elaboration and results using Origin software.

Table 2. Dimensions of structures.

Common dimensions	
Parameter	Dimensions
Length of anchors ( $L_A$ )	160 $\mu\text{m}$
Width of anchors ( $W_A$ )	20 $\mu\text{m}$
Length of central shuttle ( $L_S$ )	160 $\mu\text{m}$
Width of central shuttle ( $W_S$ )	15 $\mu\text{m}$
Length of beam ( $L_B$ )	200 $\mu\text{m}$
Width of beam ( $W_B$ )	3 $\mu\text{m}$
Dimensions of V-shape actuator	
Inclination Angle ( $\theta$ )	2.14
Dimensions of Z-shape actuator	
Length of Z-shape ( $L_Z$ -original)	3 $\mu\text{m}$
Width of Z-shape ( $W_Z$ -original)	3 $\mu\text{m}$
Dimensions of proposed Z-shape actuator	
Length of Z-shape ( $L_Z$ -modified)	1 $W_B, \dots, 66 W_B$
Width of Z-shape ( $W_Z$ -modified)	6 $\mu\text{m}$

Source: Author's own elaboration.

The simulation involves electrothermal and mechanical analysis, which consisted of the application of voltage from 0.2 V up to 5 V in steps of 0.2 V applied at one of the anchors. The other one is fixed at 0 V. Temperature was considered at 22 °C on them. The mechanical boundary conditions are given by fixing them.

Figures 5a and 5b show the voltage distribution and the displacement results, respectively, under the mentioned conditions, for the proposed Z-shape actuator.

In figure 5b, a maximum displacement of 3.31 micrometers at 5 V is shown. The temperature distribution is given in figure 5c, and the corresponding graph is shown figure 6. A maximum temperature of 274 °C is reached at 5 V.

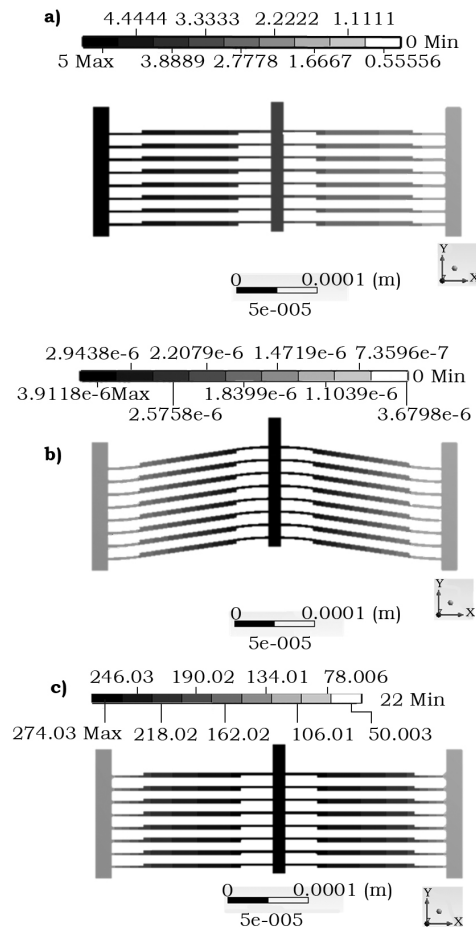


Figure 5. a) Voltage distribution for optimized Z-shape actuator. b) Displacement in the optimized Z-shape actuator. c) Temperature distribution in optimized Z-shape actuator.

Source: Author's own elaboration and results using Ansys software.

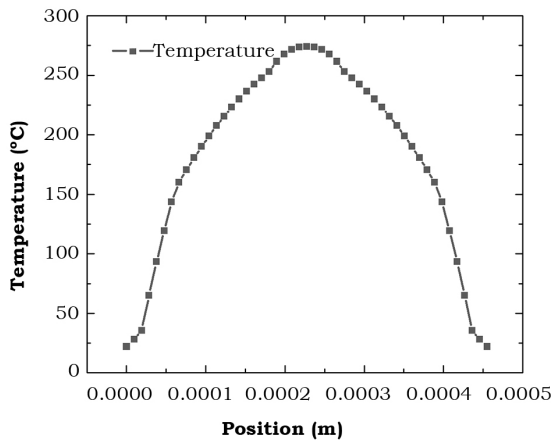


Figure 6. Temperature distribution along of the optimized Z-shape actuator. Source: Author's own elaboration and results using Origin software.

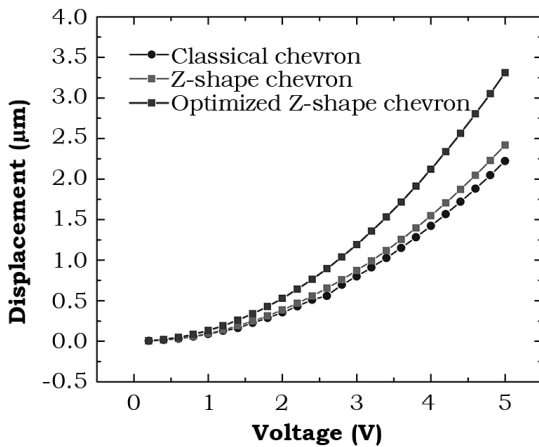


Figure 7. Displacement of the analysed geometries. Source: Author's own elaboration and results using Origin software.

In order to realize a behaviour comparison, figure 7 shows voltage-displacement curves corresponding to classical, Z-shape and optimized Z-shaped actuators are depicted. The peak displacement at 5 V in the V-shape actuator is 2.22  $\mu\text{m}$ , while in the optimized design, the displacement is of 3.31  $\mu\text{m}$ . That means, approximately 50% larger than the previous case. The maximum shuttle's displacement of the Z-shape actuator was of 2.4  $\mu\text{m}$ , which is 38% lower than the displacement obtained in the optimized case.

Figure 8 shows the variation in temperature on central part of the optimized Z-shape actuator with applied voltage. Figure 9 shows the displacements obtained by simulation and analytical calculation by means of equation 1. As it can be observed, they are almost identical.

For the case of Z-shape actuator, a similar temperature distribution as the one shown in figure 8 was observed, so it was not added here.

On table 3, a summary of the main results using geometries of eight beams is given.

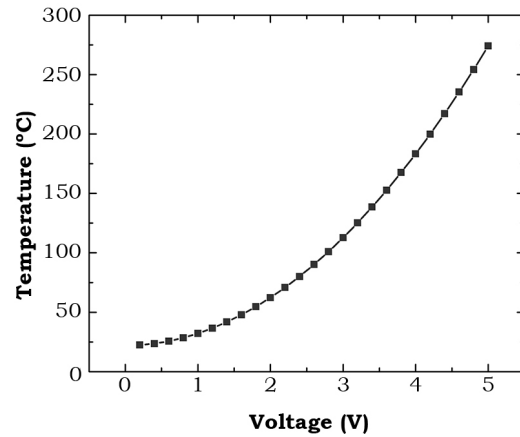


Figure 8. Temperature variation obtained using the optimized Z-shape. Source: Author's own elaboration and results using Origin software.

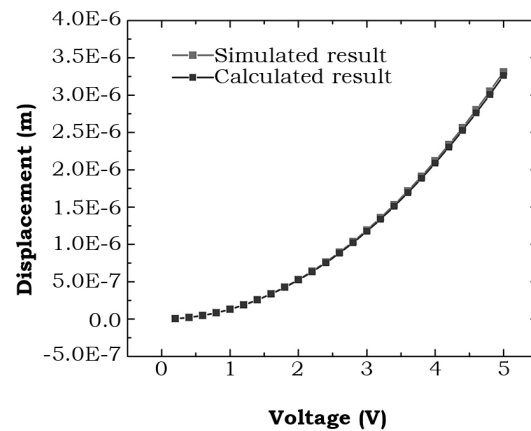


Figure 9. Simulated and calculated displacements of the optimized Z-shape. Source: Author's own elaboration and results using Origin software.

Table 3.

Obtained results.

	V-shape actuator	Z-shape actuator	Optimized Z-shape actuator
Stiffness	158.3 N/m	42.62 N/m	17.57 N/m
Output force	1092 $\mu\text{N}$	386 $\mu\text{N}$	684 $\mu\text{N}$
Displacement	2.21 $\mu\text{m}$	2.41 $\mu\text{m}$	3.31 $\mu\text{m}$

Source: Author's own elaboration.

## CONCLUSIONS

Design and simulation of an optimized electrothermal actuator with Z-shaped beams is developed. The shuttle's displacement value increases in approximately 50% compare to the V-shape actuator, and of 38% compare to the original Z-shape.

Furthermore, it can be noted that the adjusted analytical approach is extremely matched with the simulations results. As it can be observed, the total length of Z-shape of the beam is the determinant factor on the displacement.

The low stiffness of the optimized Z-shape actuator (89% lower than the original V-shape and 58% compared to Z-shape) can allow its use as load sensor.

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